

Waste Heat Recovery System by Using an Organic Rankine Cycle (ORC)

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Abstract— Though the concept of Power Plants based on the Organic Rankine Cycle (ORC) is the new technology in Bangladesh, it can play a significant role to produce power from various heat sources when other alternatives were either technically not practical or not economical. These power plants in sizes from 300 kW to 130 MW have demonstrated the maturity of this technology.^[6] The cycle is well adapted to low moderate temperature heat sources such as waste heat from industrial plants. This paper represents the feasibility of ORC based power plant in Bangladesh which has been suffering from energy crisis and unable to meet the present demand. The ORC technology is applicable to heat recovery of steel mills, rerolling mills, cement plants, and offers significant advantages over conventional steam bottoming cycles. One such system, the 2.630 MW Power Plant is now under analysis forecasts at Rahim Steel Mill in Bangladesh. [2] The environmentally friendly power plant is the first to be installed in these types of industries.^[2]

Key Words— Bangladesh, Gas Engines, Organic Rankine cycle, Organic fluid, Steam turbine, Steel Mill, Waste Heat Recovery.

1 INTRODUCTION

THERE are numerous steel plants in Bangladesh where the utilization of engine exhaust is either not possible or not required and this unused heat is lost. The ORC heat recovery system at the Rahim Steel Plant in Bangladesh is the first of such systems will be supplied to the Steel industry. This environmentally friendly plant will recover the unused engine exhaust heat and will generate 2.630 MW of electricity on a continuous basis, displacing large amount of CO₂ yearly, without interfering with the steel production process.^[1] There are 8x 3.8 MW capacity gas engines at Rahim steel where a single unit's exhaust gas flow rate 20770 kg/hr, exhaust temperature 479°C. For a single unit 1578023 kcl/hr heat will recover from this plant. Considering whole eight units, it will turn into 12624180 kcl/hr.^[1] If we will not recover the heat, the annual loss to be attributed to unused exhaust heat is million taka; a very significant expense for lost energy. The preferred approach to overcome this economically unsatisfactory situation is to use the waste heat for the generation of electrical power. In the past some steel plant operators have installed waste heat steam boilers in their plants and have utilized the process heat to operate a steam turbine generator set. The conventional steam technology has some drawbacks. In particular, with respect to stable steam turbine operation due to the high moisture content in the turbine exhaust and pinch point interference problems in the boiler. To overcome this drawback, instead of using water an organic fluid will be used such as ammonia, hydrocarbons i.e. iso-pentane, iso-octane, toluene or silicon oil, etc, instead of water.

2 HEAT RECOVERY SYSTEM FROM EXHAUST GAS, WORKING PRINCIPLE

The heat contained in the exhaust gas is transferred indirectly via a thermal oil circuit or directly to the ORC plant. The ORC plant produces electricity and low temperature heat through a closed thermodynamic cycle which follows the principle of the Organic Rankine Cycle (ORC). In the ORC process, designed as a closed cycle, the organic working medium is pre-heated in a regenerator and in a pre heater, and then vaporized through heat exchanger with the hot source. The generated vapor is expanded in a turbine that drives an asynchronous generator. Leaving the turbine, the organic working medium (still in the vapor phase) passes through the regenerator that is used to pre heat the organic liquid before vaporizing, therefore, increasing the electric efficiency through internal heat recovery. The organic vapor then condenses and delivers heat to the cooling water circuit. After the condenser, the working medium is brought back to the pressure level required (for turbine operation) by the working fluid pump and then pre-heated by internal heat exchange in the regenerator. The low temperature heat is normally discharged to a thermal user or to the atmosphere. In this specific project, as per customer request, we have considered to discharge the low temperature heat through an air condenser system.

The following blocks diagram shows the main components of the ORC based heat recovery system

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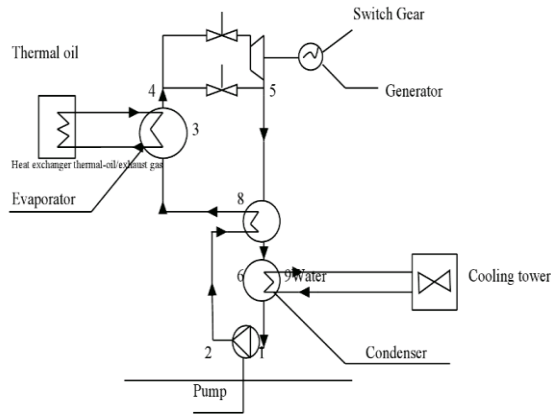


Fig.1 ORC Power plant for Engine Heat Recovery.

The operation of the ORC plant is fully automatic in normal operating conditions as well as in shut down procedures without any need of supervision personnel. In case of faulty conditions, the ORC plant will be switched off automatically and separated from the thermal oil circuit and from the electrical grid. The ORC module is normally designed to automatically adjust itself to the actual operating conditions: variations on exhaust gas temperatures and flows (in reasonable span times) will not affect the functionality of the system (but just the power output).

3 ORC versus Steam Turbine

The main difference between organic fluids and water is the lower evaporation energy of the former, so less heat is needed to evaporate the organic fluid. The evaporation of organic fluids usually takes place at lower temperature and pressure. The thermodynamic and chemical characteristics of these fluids no longer require superheating.

3.1 ORC

- Lower temperatures/pressures.
- Part load operation down to about 10%.^[3]
- High efficiency at part load.
- Simple starts and stops procedures.
- No on-site operator needed.
- No erosion of turbine blades, so need for turbine overhauls.
- Low turbine RPM.
- Low O&M requirements.

3.2 Steam Turbine

- High availability (98%).^[3]
- Higher temperatures and pressures.
- Generally NOT RECOMMENDED operate below 25% load.^[3]
- Lower part load efficiency.
- Slow start up.
- Requires on-site operator.
- High maintenance costs due to corrosion of turbine blades / casing.

- High turbine RPM.
- Higher O&M costs.
- Lower operational availability.

4 Experience with organic rankine cycle systems

The project under analysis forecasts the erection of a 2.63 MWe (gross) waste to energy power plant.^[1] As stated previously, the ORC technology could be a very efficient and flexible solution for this project. An ORC power plant is an integrated system that produces electricity from thermal energy. Fig. 2(a repeat of Fig. 1) provides a representative block diagram of an ORC power plant for engine heat recovery. Thermal energy contained in the hot engine exhaust (known as the source thermal power) is captured by a waste heat oil heater (WHOH) and transferred to the ORC turbo generator using a closed loop thermal oil subsystem.

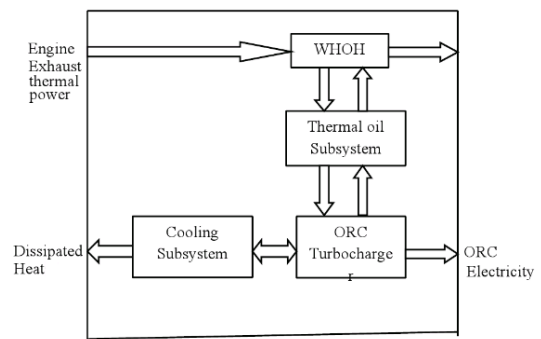


Fig. 2 ORC Power plant for Engine Heat Recovery.

The ORC turbo generator converts approximately 20% of the captured thermal power into electric power. [3] A closed loop cooling subsystem removes the balance of this thermal power from the cycle and typically dissipates it to the environment. The electrical power can be delivered to the substation to meet the customer's demand. As indicated by the quantities passing across the boundaries, the ORC power plant has three primary process interfaces:

1. Thermal power must be provided.
2. Electricity must be used.
3. Heat must be dissipated

In this project, a heat exchanger, also known as a waste heat oil heater (WHOH), is located in the thermal source stream and thermal oil is pumped through it to the turbo generator.

Thermal oil heat transfer fluids and systems are highly developed, reliable products; here we are using Exxon Mobil as thermal oil heat transfer fluid. This intermediate heat transfer approach provides great power plant layout flexibility, allowing significant separation between the thermal power source and the ORC turbo generator [more than 100 m (330 ft)].^[3] This feature is particularly beneficial for this project because multiple thermal sources (e.g., multiple engines) are in the application. Also, because the hot thermal source is not transported into the ORC turbo generator, nor is the ORC

working fluid transported into the WHOH, the working fluid is not exposed to very high temperatures and the consequent thermal stability issues. The total volume of working fluid is also minimized.

The thermal oil subsystem contains a control valve to limit the maximum input thermal energy to the level required by the ORC turbo generator to produce the rated electrical power. Signals from the ORC turbo generator to the control valve actuator command a position change to satisfy this criterion. The control valve position is returned to the ORC turbo generator controller for confirmation. If the maximum temperature of the thermal source exceeds the use temperature of available oils, a means to cool the oil (e.g. radiator) or to bypass the WHOH is required for periods when the thermal source is flowing but the turbo generator is off line (i.e. not removing heat from the oil). Considering the characteristics of the forecasted power plant, the suitable ORC solution is the employment of one Turboden 30 –HR is being selected; in the table 1 the performance of the unit, data for designing of Waste Heat Recovery Exchanger Thermal Oil Heater are given.

Table 1. ^[1-3]

Description	Value	Unit
Location	Bangladesh	-
Design ambient Air temperature	35	°C
Waste Heat Details:	-	-
Primary heat source	Exhaust gas from engines	-
Engines Model	MWM TCG2032V16	-
Number of engines employed	8	-
Exhaust Gas Mass Flow from 1 No. unit	5.77	(kg/s)
Exhaust Gas Mass Flow from 1 No. unit	20770	(kg/hr)
Exhaust Gas Mass Flow from 1 No. unit	20770	(kg/hr)
Exhaust Gas Mass Flow from 8 Nos. Unit	166160	(kg/hr)
Exhaust gas temperature	479	°C
Outlet temperature Restriction	190	°C
Hot oil Temperature at HEX Inlet	160	°C
Hot oil Temperature at HEX Outlet	280	°C
Total Hot Oil Flow from 1 No. HEX	24312	(kg/hr)
Total Hot Oil Flow from 8 Nos. HEX	194493	(kg/hr)
Net Total Heat Recovery In 1 No. HEX	1578023	(kcal/hr)
Net Total Heat Recovery In 8 Nos. HEX	12624180	(kcal/hr)

Net Total Heat Recovery In 8 Nos. HEX	14.68	(MW)
Allowable pressure Drop Across HEX & Diverter valve	500	(mmWC)
SHC of the heat source	1,116	Kj/kgk
Heat source cool down to	170	Deg.C
Thermal loss consider	1%	-
Overall thermal power recover	15.752	MWt
ORC units employable	Turboden 30-HR	-
ORC hot hot source	Thermal oil	-
ORC thermal power input	15.752	MWt
Hot source nominal temperature in/out [+/- 5°C]	285/145	°C
Cooling water temperature in/out [+/- 5°C]	42/60	°C
Exhaust Gas Composition (V/V%) customer to confirm in writing	Engine Exhaust gas	
CO2	8	%
O2	10.1	%
H2O	4.8	%
N2	77.1	%
Gross ORC active electric power [+/- 10°C]	2.63	MWe
ORC captive consumption(*)	130	kWe
Net ORC active electric power output [+/- 10%]	2500	kWe
Electric generator	Asynchronous, 3 phase, 50 Hz, M.V.	-

(*) Net of ORC internal consumption (mainly feed pump). No external consumption are included (i.e. thermal oil system, heat dissipation system, etc).

5 Results

By considering thermal losses 1% for the eight units, the gross ORC active electric power can be achieved (+/-10%) 2.630 MWe.^[1]

6 Conclusion

For the cogeneration of industrial waste, the use of ORC offers some potentiality. As the cost of energy increases day by day, the use of organic Rankine cycles for effective energy utilization can be expected to become more important in developing

countries like Bangladesh. ORC has demonstrated advantages over conventional steam turbine and large amount of power can be achieved in gas engine power plants through enhanced waste-heat recovery while providing distinct cost and environmental advantages.

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